

**PHY 711 Classical Mechanics and  
Mathematical Methods**

**10-10:50 AM MWF in Olin 103**

**Discussion on Lecture 17: Chap. 4 (F&W)**

**Normal Mode Analysis**

- 1. Normal modes for extended one-dimensional systems**
- 2. Normal modes for 2 and 3 dimensional systems**

9/29/2021

PHY 711 Fall 2021 – Lecture 17

1

In this lecture, we will extend our normal mode analysis to more complicated systems, including infinite periodic systems and beyond one dimension.

# PHYSICS COLLOQUIUM

THURSDAY  
•  
SEPTEMBER 30, 2021

## Part III Theoretical and Computational Projects

Professor Stephen Winter, representing theoretical and computational research projects in the physics department of Wake Forest University, include the focus areas of his group and that of Professor Timo Thonhauser. Research efforts by Natalie Holzwarth and William C. Kerr will also be discussed. This presentation will complete the three part snapshot of research opportunities at WFU Physics.

## Discussion on Colloquium Program

This second part of the colloquium will be devoted to a discussion of the physics colloquium series, exchanging ideas with the goal of improvement and optimization. Bring your thoughts and ideas to the discussion.

Wake Forest University  
Physics Department  
Research Opportunities  
and Discussion on  
Colloquium Program

4:00 pm - Olin 101

Note: For additional information on the seminar or to obtain the video conference link, contact [wfuphys@wfu.edu](mailto:wfuphys@wfu.edu)

Reception at 3:30pm - Olin Lounge

\*We encourage all to wander outside to the front entrance or up to the Observatory Deck on the 3<sup>rd</sup> floor to enjoy their refreshments.

9/29/2021

2

## Course schedule

(Preliminary schedule -- subject to frequent adjustment.)

	Date	F&W Reading	Topic	Assignment	Due
1	Mon, 8/23/2021	Chap. 1	Introduction	<a href="#">#1</a>	8/27/2021
2	Wed, 8/25/2021	Chap. 1	Scattering theory	<a href="#">#2</a>	8/30/2021
3	Fri, 8/27/2021	Chap. 1	Scattering theory		
4	Mon, 8/30/2021	Chap. 1	Scattering theory	<a href="#">#3</a>	9/01/2021
5	Wed, 9/01/2021	Chap. 1	Summary of scattering theory	<a href="#">#4</a>	9/03/2021
6	Fri, 9/03/2021	Chap. 2	Non-inertial coordinate systems	<a href="#">#5</a>	9/06/2021
7	Mon, 9/06/2021	Chap. 3	Calculus of Variation	<a href="#">#6</a>	9/10/2021
8	Wed, 9/08/2021	Chap. 3	Calculus of Variation		
9	Fri, 9/10/2021	Chap. 3 & 6	Lagrangian Mechanics	<a href="#">#7</a>	9/13/2021
10	Mon, 9/13/2021	Chap. 3 & 6	Lagrangian Mechanics	<a href="#">#8</a>	9/17/2021
11	Wed, 9/15/2021	Chap. 3 & 6	Constants of the motion		
12	Fri, 9/17/2021	Chap. 3 & 6	Hamiltonian equations of motion	<a href="#">#9</a>	9/20/2021
13	Mon, 9/20/2021	Chap. 3 & 6	Liouville theorem	<a href="#">#10</a>	9/22/2021
14	Wed, 9/22/2021	Chap. 3 & 6	Canonical transformations		
15	Fri, 9/24/2021	Chap. 4	Small oscillations about equilibrium	<a href="#">#11</a>	9/27/2021
16	Mon, 9/27/2021	Chap. 4	Normal modes of vibration	<a href="#">#12</a>	9/29/2021
17	Wed, 9/29/2021	Chap. 4	Normal modes of more complicated systems	<a href="#">#13</a>	10/04/2021
18	Fri, 10/01/2021	Chap. 7	Motion of strings		

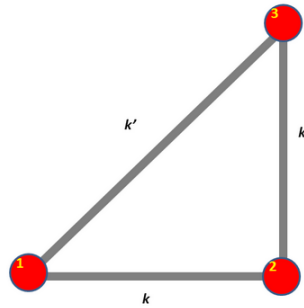


This is the last lecture for Chap. 4. On Friday we will continue to discuss vibrations in extended one dimensional motion as covered in Chap. 7.

## PHY 711 -- Assignment #13

Sept. 29, 2021

Finish reading Chapter 4 in Fetter & Walecka.



1. Consider the system of 3 masses ( $m_1=m_2=m_3=m$ ) shown attached by elastic forces in the right triangular configuration (with angles 45, 90, 45 deg) shown above with spring constants  $k$  and  $k'$ . Find the normal modes of small oscillations for this system. For numerical evaluation, you may assume that  $k=k'$ .

9/29/2021

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4

Homework due Monday.

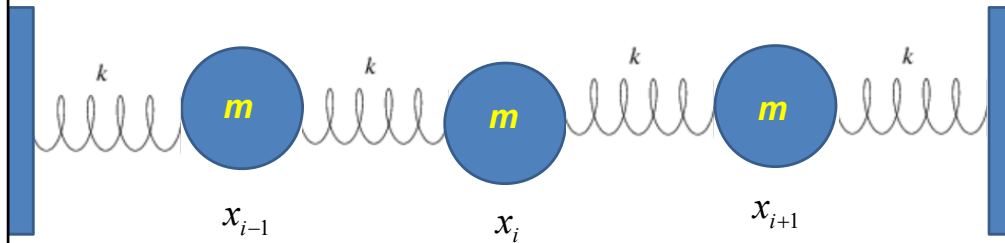
Your questions –

From Owen -- If a spring system were undergoing chaotic motion, would the eigenvalue/eigenvector formalism we are discussing still apply? Can one know in advance whether a system will behave chaotically or not?

Comment – In the present treatment we are focusing on the linearized equations of motion. In this case, while the motion can be complicated (superposition of several modes for example), the chaotic behavior does not occur.

Mathematically, chaotic behavior may occur in the presence of non-linear contributions. Physically, non-linear contributions tend to be important when the system has large deviations from equilibrium.

Consider an extended system of masses and springs:



Note: each mass coordinate is measured relative to its equilibrium position  $x_i^0$

$$L = T - V = \frac{1}{2}m \sum_{i=1}^N \dot{x}_i^2 - \frac{1}{2}k \sum_{i=0}^N (x_{i+1} - x_i)^2$$

Note: In fact, we have  $N$  masses;  $x_0$  and  $x_{N+1}$  will be treated using boundary conditions.

9/29/2021

PHY 711 Fall 2021 – Lecture 17

6

Example of one dimensional system with fixed boundary values.

$$L = T - V = \frac{1}{2} m \sum_{i=1}^N \dot{x}_i^2 - \frac{1}{2} k \sum_{i=0}^N (x_{i+1} - x_i)^2$$

$$x_0 \equiv 0 \text{ and } x_{N+1} \equiv 0$$

From Euler - Lagrange equations :

$$m\ddot{x}_1 = k(x_2 - 2x_1)$$

$$m\ddot{x}_2 = k(x_3 - 2x_2 + x_1)$$

.....

$$m\ddot{x}_i = k(x_{i+1} - 2x_i + x_{i-1})$$

.....

$$m\ddot{x}_N = k(x_{N-1} - 2x_N)$$

Review of detailed equations.

From Euler - Lagrange equations :

$$m\ddot{x}_j = k(x_{j+1} - 2x_j + x_{j-1}) \quad \text{with } x_0 = 0 = x_{N+1}$$

Try:  $x_j(t) = Ae^{-i\omega t + iqa_j}$

$$-\omega^2 Ae^{-i\omega t + iqa_j} = \frac{k}{m}(e^{iqa} - 2 + e^{-iqa})Ae^{-i\omega t + iqa_j}$$

$$-\omega^2 = \frac{k}{m}(2\cos(qa) - 2)$$

$$\Rightarrow \omega^2 = \frac{4k}{m}\sin^2\left(\frac{qa}{2}\right)$$

Review of solutions discussed on Wednesday.



From Euler - Lagrange equations - - continued :

$$m\ddot{x}_j = k(x_{j+1} - 2x_j + x_{j-1}) \quad \text{with } x_0 = 0 = x_{N+1}$$

$$\text{Try: } x_j(t) = Ae^{-i\omega t + iqa_j} \quad \Rightarrow \omega^2 = \frac{4k}{m} \sin^2\left(\frac{qa}{2}\right)$$

$$\text{Note that: } x_j(t) = Be^{-i\omega t - iqa_j} \quad \Rightarrow \omega^2 = \frac{4k}{m} \sin^2\left(\frac{qa}{2}\right)$$

General solution :

$$x_j(t) = \Re\left(Ae^{-i\omega t + iqa_j} + Be^{-i\omega t - iqa_j}\right)$$

Impose boundary conditions :

$$x_0(t) = \Re\left(Ae^{-i\omega t} + Be^{-i\omega t}\right) = 0$$

$$x_{N+1}(t) = \Re\left(Ae^{-i\omega t + iqa(N+1)} + Be^{-i\omega t - iqa(N+1)}\right) = 0$$

Review of boundary conditions.

Impose boundary conditions -- continued:

$$x_0(t) = \Re\left(Ae^{-i\omega t} + Be^{-i\omega t}\right) = 0$$

$$x_{N+1}(t) = \Re\left(Ae^{-i\omega t + iqa(N+1)} + Be^{-i\omega t - iqa(N+1)}\right) = 0$$

$$\Rightarrow B = -A$$

$$x_{N+1}(t) = \Re\left(Ae^{-i\omega t} \left(e^{iqa(N+1)} - e^{-iqa(N+1)}\right)\right) = 0$$

$$\Rightarrow \sin(qa(N+1)) = 0$$

$$\Rightarrow qa(N+1) = \nu\pi \quad \text{where } \nu = 1, 2, \dots, N$$

$$qa = \frac{\nu\pi}{N+1}$$

Recap -- solution for integer parameter  $\nu$

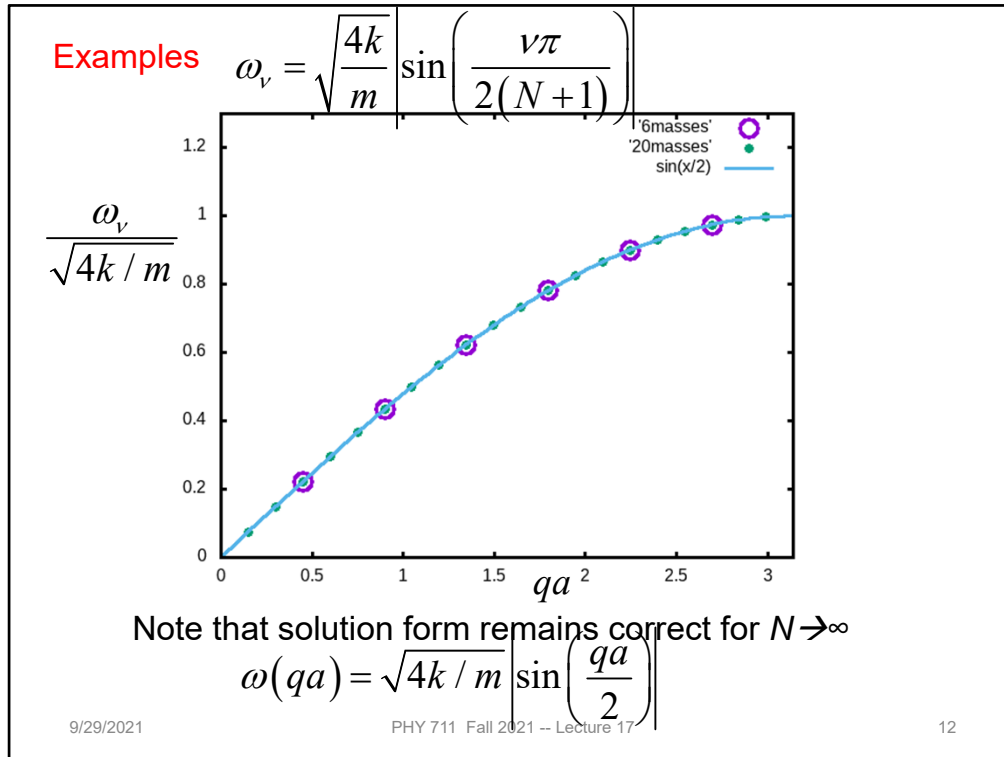
$$x_j(t) = \Re \left( 2iAe^{-i\omega_\nu t} \sin \left( \frac{\nu \pi j}{N+1} \right) \right)$$

$$\omega_\nu^2 = \frac{4k}{m} \sin^2 \left( \frac{\nu \pi}{2(N+1)} \right)$$

Note that non - trivial, unique values are

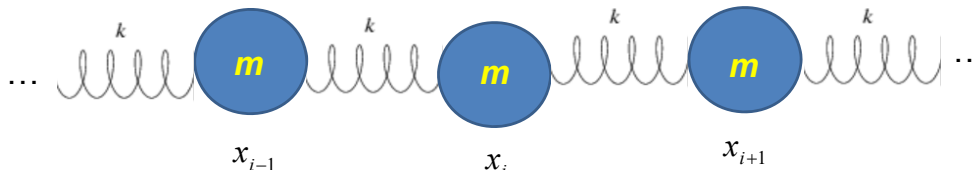
$$\nu = 1, 2, \dots, N$$

Review of full solution.



Plot for example. Now consider the case where N is very large.

For extended chain without boundaries:



From Euler-Lagrange equations:

$$m\ddot{x}_j = k(x_{j+1} - 2x_j + x_{j-1}) \quad \text{for all } x_j$$

Try:  $x_j(t) = Ae^{-i\omega t + iqa_j}$

$$-\omega^2 Ae^{-i\omega t + iqa_j} = \frac{k}{m}(e^{iqa} - 2 + e^{-iqa})Ae^{-i\omega t + iqa_j}$$

$$-\omega^2 = \frac{k}{m}(2\cos(qa) - 2)$$

$$\Rightarrow \omega^2 = \frac{4k}{m}\sin^2\left(\frac{qa}{2}\right) \quad \text{distinct values for } 0 \leq qa \leq \pi$$

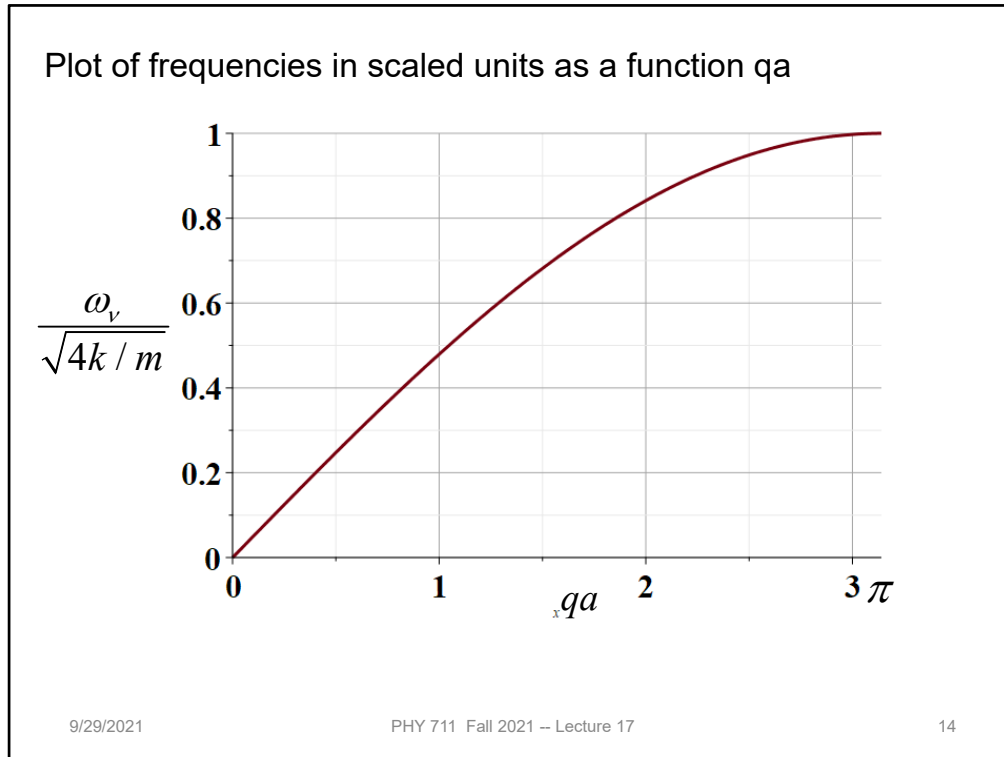
**Note that we are assuming that all masses and springs are identical here.**

9/29/2021

PHY 711 Fall 2021 – Lecture 17

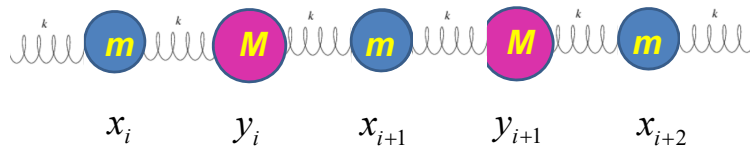
13

Now consider the case where  $N$  is infinite so that there are an infinite number of solutions parameterized by  $qa$  as a continuous variable.



Distinct solutions occur for  $qa$  in the range of 0- $\pi$  as shown in the plot.

Consider an infinite system of masses and springs now with two kinds of masses:



Note: each mass coordinate is measured relative to its equilibrium position  $x_i^0 \equiv 0, y_i^0 \equiv 0, \dots$

$$L = T - V$$

$$= \frac{1}{2} m \sum_{i=0}^{\infty} \dot{x}_i^2 + \frac{1}{2} M \sum_{i=0}^{\infty} \dot{y}_i^2 - \frac{1}{2} k \sum_{i=0}^{\infty} (x_{i+1} - y_i)^2 - \frac{1}{2} k \sum_{i=0}^{\infty} (y_i - x_i)^2$$

9/29/2021

PHY 711 Fall 2021 – Lecture 17

15

Now consider a slight modification of the previous example where masses are alternately  $m$  and  $M$  with labels  $x$  and  $y$ .

$$L = T - V$$

$$= \frac{1}{2} m \sum_{i=0}^{\infty} \dot{x}_i^2 + \frac{1}{2} M \sum_{i=0}^{\infty} \dot{y}_i^2 - \frac{1}{2} k \sum_{i=0}^{\infty} (x_{i+1} - y_i)^2 - \frac{1}{2} k \sum_{i=0}^{\infty} (y_i - x_i)^2$$

Euler - Lagrange equations :

$$m\ddot{x}_j = k(y_{j-1} - 2x_j + y_j)$$

$$M\ddot{y}_j = k(x_j - 2y_j + x_{j+1})$$

Trial solution :

$$x_j(t) = A e^{-i\omega t + i2qa_j}$$

Note that  $2qa$  is an unknown parameter.

$$y_j(t) = B e^{-i\omega t + i2qa_j}$$

Does this form seem reasonable?

$$\begin{pmatrix} m\omega^2 - 2k & k(e^{-i2qa} + 1) \\ k(e^{i2qa} + 1) & M\omega^2 - 2k \end{pmatrix} \begin{pmatrix} A \\ B \end{pmatrix} = 0$$

9/29/2021

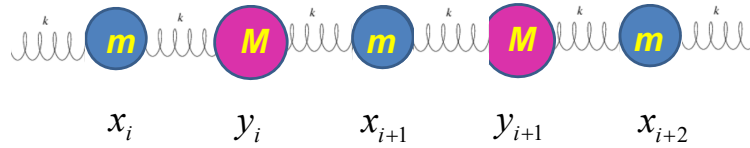
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16

In this case, we can analyze the system by considering different amplitudes for the  $m$  and  $M$  masses. The resulting coupled equations can be written in matrix form.



Comment on notation --



Trial solution:

$$x_j(t) = Ae^{-i\omega t + i2qaj}$$

$$y_j(t) = Be^{-i\omega t + i2qaj}$$

*Using  $2qa$  as our unknown parameter is a convenient choice so that we can easily relate our solution to the  $m=M$  case.*

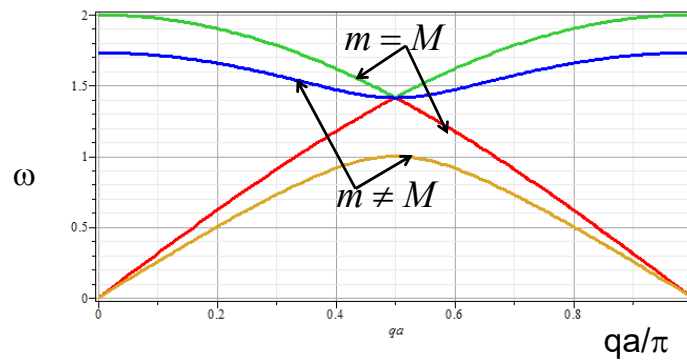
$$\begin{pmatrix} m\omega^2 - 2k & k(e^{-i2qa} + 1) \\ k(e^{i2qa} + 1) & M\omega^2 - 2k \end{pmatrix} \begin{pmatrix} A \\ B \end{pmatrix} = 0$$

Solutions :

$$\omega_{\pm}^2 = \frac{k}{m} + \frac{k}{M} \pm k \sqrt{\frac{1}{m^2} + \frac{1}{M^2} + \frac{2\cos(2qa)}{mM}}$$

Note that for  $m=M$ , we obtain the same normal modes as before. Is this reassuring?

- a. No
- b. Yes



9/29/2021

PHY 711 Fall 2021 – Lecture 17

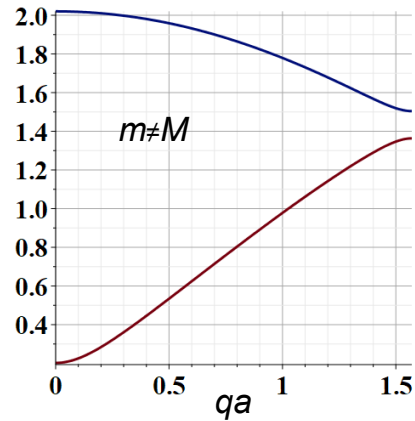
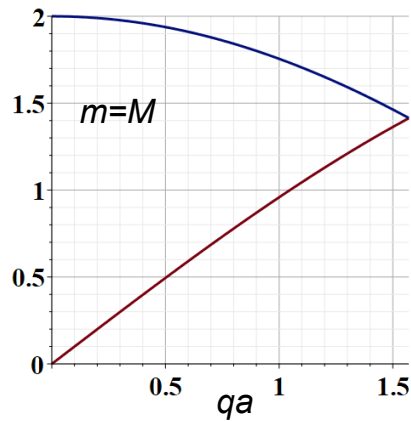
18

Plotting the solutions for the frequencies as a function of  $qa$ .

Normal mode frequencies:

$$\omega_{\pm}^2 = \frac{k}{m} + \frac{k}{M} \pm k \sqrt{\frac{1}{m^2} + \frac{1}{M^2} + \frac{2 \cos(2qa)}{mM}}$$

Note that for every  $qa$ , there are 2 modes.



Plotting only distinct frequencies  $0 < qa < \pi/2$

Eigenvectors:

For  $qa = 0$ :

$$\omega_- = 0 \qquad \omega_+ = \sqrt{\frac{2k}{m} + \frac{2k}{M}}$$

$$\begin{pmatrix} A \\ B \end{pmatrix}_- = N \begin{pmatrix} 1 \\ 1 \end{pmatrix} \qquad \begin{pmatrix} A \\ B \end{pmatrix}_+ = N \begin{pmatrix} 1 \\ -1 \end{pmatrix}$$

For  $qa = \frac{\pi}{2}$ :

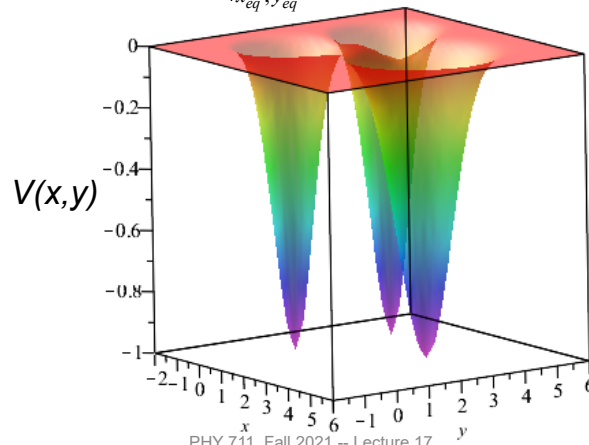
$$\omega_- = \sqrt{\frac{2k}{M}} \qquad \omega_+ = \sqrt{\frac{2k}{m}}$$

$$\begin{pmatrix} A \\ B \end{pmatrix}_- = N \begin{pmatrix} 1 \\ 0 \end{pmatrix} \qquad \begin{pmatrix} A \\ B \end{pmatrix}_+ = N \begin{pmatrix} 0 \\ 1 \end{pmatrix}$$

Some details about the solutions.

### Potential in 2 and more dimensions

$$V(x, y) \approx V(x_{eq}, y_{eq}) + \frac{1}{2}(x - x_{eq})^2 \left. \frac{\partial^2 V}{\partial x^2} \right|_{x_{eq}, y_{eq}} + \frac{1}{2}(y - y_{eq})^2 \left. \frac{\partial^2 V}{\partial y^2} \right|_{x_{eq}, y_{eq}} + (x - x_{eq})(y - y_{eq}) \left. \frac{\partial^2 V}{\partial x \partial y} \right|_{x_{eq}, y_{eq}}$$



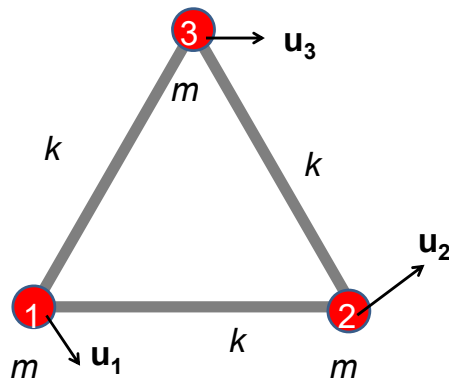
9/29/2021

PHY 711 Fall 2021 – Lecture 17

21

Returning to the finite systems, consider equilibria in two dimensions as shown.

Example – normal modes of a system with the symmetry of an equilateral triangle



Degrees of freedom for  
2-dimensional motion:  
 $2N = 6$

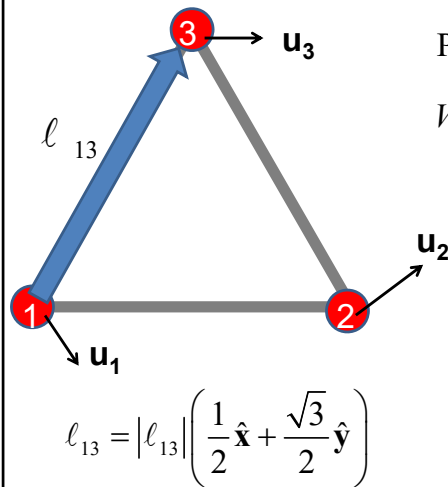
9/29/2021

PHY 711 Fall 2021 – Lecture 17

22

Specifically, we will consider 3 masses in an equilateral triangle configuration as shown.

Example – normal modes of a system with the symmetry of an equilateral triangle -- continued



Potential contribution for spring 13:

$$\begin{aligned} V_{13} &= \frac{1}{2} k \left( |\ell_{13} + \mathbf{u}_3 - \mathbf{u}_1| - |\ell_{13}| \right)^2 \\ &\approx \frac{1}{2} k \left( \frac{\ell_{13} \cdot (\mathbf{u}_3 - \mathbf{u}_1)}{|\ell_{13}|} \right)^2 \\ &\approx \frac{1}{2} k \left( \frac{1}{2} (u_{x3} - u_{x1}) + \frac{\sqrt{3}}{2} (u_{y3} - u_{y1}) \right)^2 \end{aligned}$$

9/29/2021

PHY 711 Fall 2021 -- Lecture 17

23

We need to consider displacements from equilibrium in the x-y plane. Keeping only linear terms in the displacements we wind up with a simple relationship to analyze.

Some details for spring 13:

$$\left(|\ell_{13} + \mathbf{u}_3 - \mathbf{u}_1| - |\ell_{13}|\right)^2 \equiv \left((\ell_{13} + \mathbf{u}_{13})^{1/2} - |\ell_{13}|\right)^2$$

$$(\ell_{13} + \mathbf{u}_{13})^{1/2} = |\ell_{13}| \left(1 + \frac{2\ell_{13} \cdot \mathbf{u}_{13}}{|\ell_{13}|^2} + \frac{|\mathbf{u}_{13}|^2}{|\ell_{13}|^2}\right)^{1/2} \quad \text{Assume } |\mathbf{u}_{13}| \ll |\ell_{13}|$$

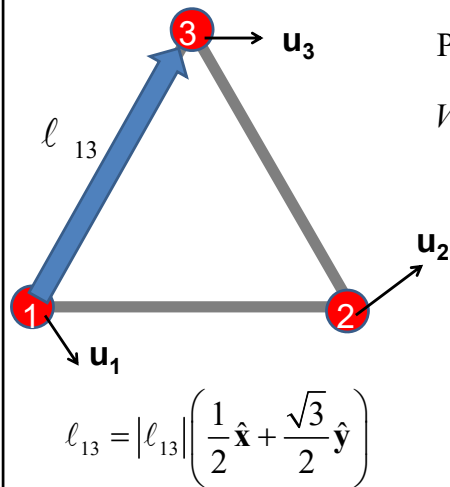
$$\approx |\ell_{13}| \left(1 + \frac{\ell_{13} \cdot \mathbf{u}_{13}}{|\ell_{13}|^2}\right) = |\ell_{13}| + \frac{\ell_{13} \cdot \mathbf{u}_{13}}{|\ell_{13}|}$$

$$\Rightarrow \left((\ell_{13} + \mathbf{u}_{13})^{1/2} - |\ell_{13}|\right)^2 = \left(\frac{\ell_{13} \cdot \mathbf{u}_{13}}{|\ell_{13}|}\right)^2$$

**Note that this analysis of the leading term is true in 1, 2, and 3 dimensions.**



Example – normal modes of a system with the symmetry of an equilateral triangle -- continued



Potential contribution for spring 13:

$$\begin{aligned} V_{13} &= \frac{1}{2} k \left( |\ell_{13} + \mathbf{u}_3 - \mathbf{u}_1| - |\ell_{13}| \right)^2 \\ &\approx \frac{1}{2} k \left( \frac{\ell_{13} \cdot (\mathbf{u}_3 - \mathbf{u}_1)}{|\ell_{13}|} \right)^2 \\ &\approx \frac{1}{2} k \left( \frac{1}{2} (u_{x3} - u_{x1}) + \frac{\sqrt{3}}{2} (u_{y3} - u_{y1}) \right)^2 \end{aligned}$$

9/29/2021

PHY 711 Fall 2021 -- Lecture 17

25

We need to consider displacements from equilibrium in the x-y plane. Keeping only linear terms in the displacements we wind up with a simple relationship to analyze.

Example – normal modes of a system with the symmetry of an equilateral triangle -- continued

Potential contributions:  $V = V_{12} + V_{13} + V_{23}$

$$\begin{aligned} &\approx \frac{1}{2}k \left( \frac{\ell_{12} \cdot (\mathbf{u}_2 - \mathbf{u}_1)}{|\ell_{12}|} \right)^2 + \frac{1}{2}k \left( \frac{\ell_{13} \cdot (\mathbf{u}_3 - \mathbf{u}_1)}{|\ell_{13}|} \right)^2 \\ &\quad + \frac{1}{2}k \left( \frac{\ell_{23} \cdot (\mathbf{u}_3 - \mathbf{u}_2)}{|\ell_{23}|} \right)^2 \\ &\approx \frac{1}{2}k (u_{x2} - u_{x1})^2 \\ &\quad + \frac{1}{2}k \left( \frac{1}{2}(u_{x3} - u_{x1}) + \frac{\sqrt{3}}{2}(u_{y3} - u_{y1}) \right)^2 \\ &\quad + \frac{1}{2}k \left( \frac{1}{2}(u_{x2} - u_{x3}) - \frac{\sqrt{3}}{2}(u_{y2} - u_{y3}) \right)^2 \end{aligned}$$

9/29/2021

PHY 711 Fall 2021 – Lecture 17

26

Analyzing the 3 displacements for the equilateral triangle geometry, we find these equations.

Example – normal modes of a system with the symmetry of an equilateral triangle -- continued

$$\frac{k}{m} \begin{bmatrix} \frac{5}{4} & -1 & -\frac{1}{4} & \frac{1}{4}\sqrt{3} & 0 & -\frac{1}{4}\sqrt{3} \\ -1 & \frac{5}{4} & -\frac{1}{4} & 0 & -\frac{1}{4}\sqrt{3} & \frac{1}{4}\sqrt{3} \\ -\frac{1}{4} & -\frac{1}{4} & \frac{1}{2} & -\frac{1}{4}\sqrt{3} & \frac{1}{4}\sqrt{3} & 0 \\ \frac{1}{4}\sqrt{3} & 0 & -\frac{1}{4}\sqrt{3} & \frac{3}{4} & 0 & -\frac{3}{4} \\ 0 & -\frac{1}{4}\sqrt{3} & \frac{1}{4}\sqrt{3} & 0 & \frac{3}{4} & -\frac{3}{4} \\ -\frac{1}{4}\sqrt{3} & \frac{1}{4}\sqrt{3} & 0 & -\frac{3}{4} & -\frac{3}{4} & \frac{3}{2} \end{bmatrix} \begin{bmatrix} u_{x1} \\ u_{x2} \\ u_{x3} \\ u_{y1} \\ u_{y2} \\ u_{y3} \end{bmatrix} = \omega^2 \begin{bmatrix} u_{x1} \\ u_{x2} \\ u_{x3} \\ u_{y1} \\ u_{y2} \\ u_{y3} \end{bmatrix}$$

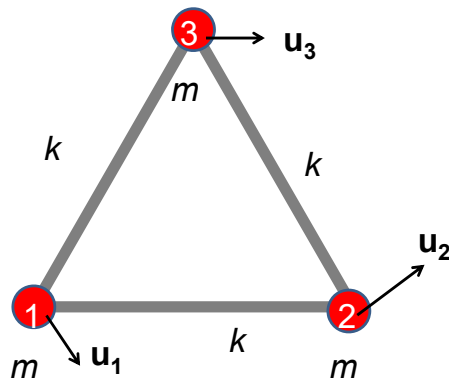
9/29/2021

PHY 711 Fall 2021 -- Lecture 17

27

The results is a 6x6 matrix problem to find eigenvalues and eigenvectors.

Example – normal modes of a system with the symmetry of an equilateral triangle -- continued



With help from Maple

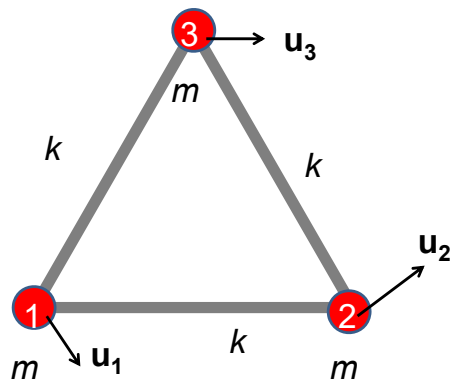
$$\omega^2 = \begin{bmatrix} 3 \\ \frac{3}{2} \\ \frac{3}{2} \\ 0 \\ 0 \\ 0 \end{bmatrix} \frac{k}{m}$$

9/29/2021

PHY 711 Fall 2021 -- Lecture 17

28

Results from Maple. We have 6 eigenvalues and 3 non-zero modes for this case.



What can you say about the 3 zero frequency modes?

What can you say about the 3 non-zero frequency modes?

### 3-dimensional periodic lattices

Example – face-centered-cubic unit cell (Al or Ni)

Diagram of  
atom positions

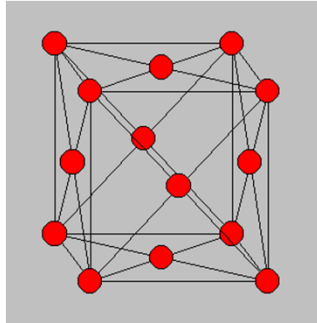
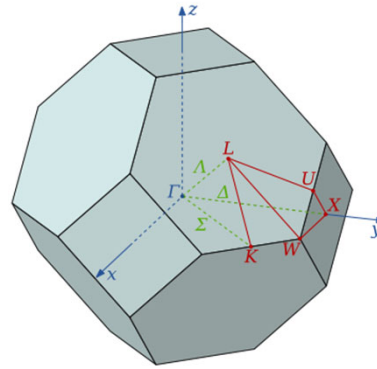


Diagram of q-  
space  $\nu(q)$



9/29/2021

PHY 711 Fall 2021 -- Lecture 17

30

Interesting extensions to a 3-dimensional crystalline system.

From: PRB **59** 3395 (1999); Mishin et. al.  $\nu(q)$

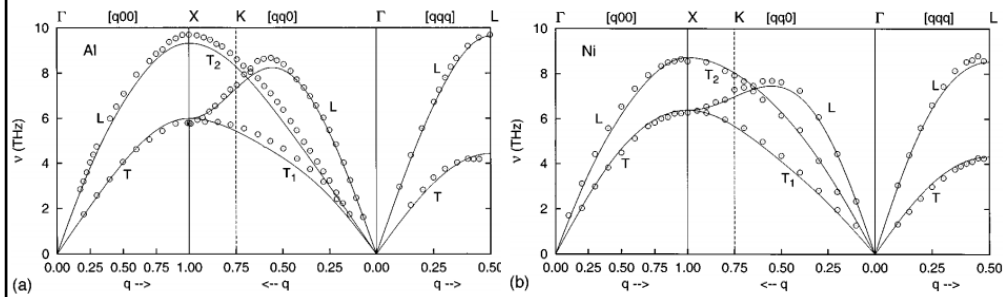


FIG. 2. Comparison of phonon-dispersion curves for Al (a) and Ni (b) predicted by the present EAM potentials, with the experimental values measured by neutron diffraction at 80 K (Al) and 298 K (Ni) (Ref. 33 for Al and Ref. 34 for Ni). The phonon frequencies at point  $X$  were included in the fitting database with low weight.

Note that for each  $q$ , there are 3 frequencies.

9/29/2021

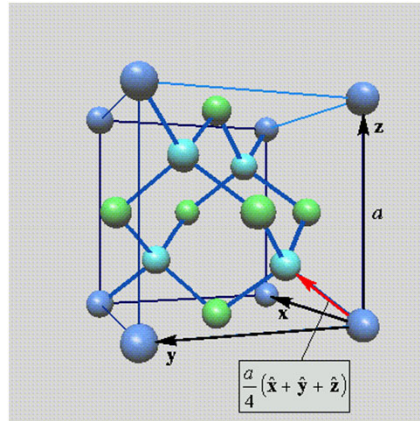
PHY 711 Fall 2021 – Lecture 17

31

Results of normal modes from experiment and simulations for face centered cubic Al (left) and Ni (right). Interestingly, the phonon frequency patterns are similar for these very different materials.

## Lattice vibrations for 3-dimensional lattice

Example: diamond lattice



Ref: [http://phycomp.technion.ac.il/~nika/diamond\\_structure.html](http://phycomp.technion.ac.il/~nika/diamond_structure.html)

9/29/2021

PHY 711 Fall 2021 -- Lecture 17

32

Another example – diamond.



Atoms located at the positions :

$$\mathbf{R}^a = \mathbf{R}_0^a + \mathbf{u}^a$$

Potential energy function near equilibrium :

$$U(\{\mathbf{R}^a\}) \approx U(\{\mathbf{R}_0^a\}) + \frac{1}{2} \sum_{a,b} (\mathbf{R}^a - \mathbf{R}_0^a) \cdot \left. \frac{\partial^2 U}{\partial \mathbf{R}^a \partial \mathbf{R}^b} \right|_{\{\mathbf{R}_0^a\}} \cdot (\mathbf{R}^b - \mathbf{R}_0^b)$$

Define :

$$D_{jk}^{ab} \equiv \left. \frac{\partial^2 U}{\partial \mathbf{R}_j^a \partial \mathbf{R}_k^b} \right|_{\{\mathbf{R}_0^a\}}$$

so that

$$U(\{\mathbf{R}^a\}) \approx U_0 + \frac{1}{2} \sum_{a,b,j,k} u_j^a D_{jk}^{ab} u_k^b$$

$$L(\{\mathbf{u}_j^a, \dot{\mathbf{u}}_j^a\}) = \frac{1}{2} \sum_{a,j} m_a (\dot{\mathbf{u}}_j^a)^2 - U_0 - \frac{1}{2} \sum_{a,b,j,k} u_j^a D_{jk}^{ab} u_k^b$$

Some equations for extended systems.

$$L(\{u_j^a, \dot{u}_j^a\}) = \frac{1}{2} \sum_{a,j} m_a (\dot{u}_j^a)^2 - U_0 - \frac{1}{2} \sum_{a,b,j,k} u_j^a D_{jk}^{ab} u_k^b$$

Equations of motion :

$$m_a \ddot{u}_j^a = - \sum_{b,k} D_{jk}^{ab} u_k^b$$

Solution form :

$$u_j^a(t) = \frac{1}{\sqrt{m_a}} A_j^a e^{-i\omega t + i\mathbf{q} \cdot \mathbf{R}_0^a}$$

Details:  $\mathbf{R}_0^a = \boldsymbol{\tau}^a + \mathbf{T}$  where  $\boldsymbol{\tau}^a$  denotes  
unique sites and  
 $\mathbf{T}$  denotes replicas

More equations.

Define :

$$W_{jk}^{ab}(\mathbf{q}) = \sum_{\mathbf{T}} \frac{D_{jk}^{ab} e^{i\mathbf{q} \cdot (\mathbf{r}^a - \mathbf{r}^b)}}{\sqrt{m_a m_b}} e^{i\mathbf{q} \cdot \mathbf{T}}$$

Eigenvalue equations :

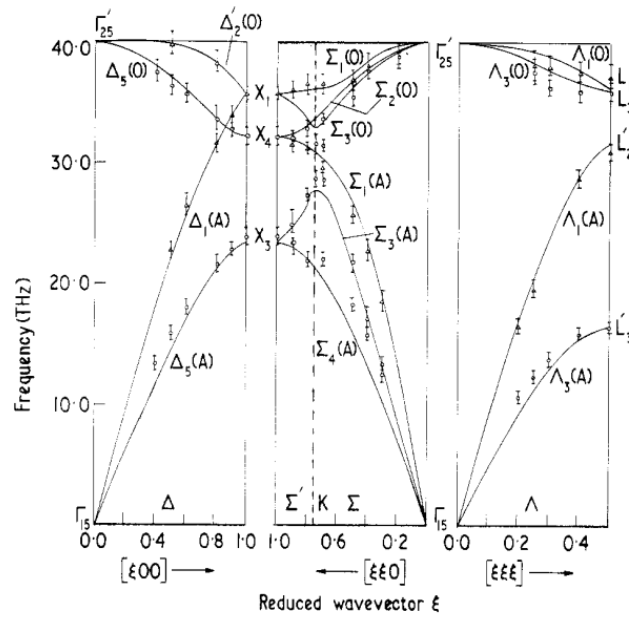
$$\omega^2 A_j^a = \sum_{b,k} W(\mathbf{q})_{jk}^{ab} A_k^b$$

In this equation the summation is only over unique atomic sites.

$\Rightarrow$  Find "dispersion curves"  $\omega(\mathbf{q})$

More equations.

B. P. Pandey and B. Dayal, J. Phys. C. Solid State Phys. **6** 2943 (1973)



**Figure 2.** Phonon dispersion curves of diamond. Experimental points *et al* (1965, 1967).  $\Delta$  and  $\circ$  represent the longitudinal and transverse modes.

9/29/2021

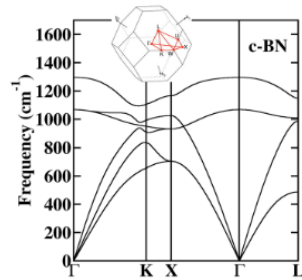
PHY 711 Fall 2021 – Lecture 17

36

Results for diamond from simulation and experiment.

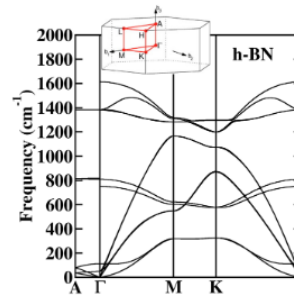
## Examples of phonon spectra of two forms of boron nitride

### Cubic structure



**Figure 1.** Phonon dispersion curves ( $\omega^{\nu}(\mathbf{q})$ ) for cubic BN. The inset Brillouin zone diagram was reprinted from Setyawan *et al* [7], copyright (2010), with permission from Elsevier.

### Hexagonal structure



**Figure 2.** Phonon dispersion curves ( $\omega^{\nu}(\mathbf{q})$ ) for hexagonal BN. The inset Brillouin zone diagram was reprinted from Setyawan *et al* [7], copyright (2010), with permission from Elsevier.