



PHY 711 Classical Mechanics and Mathematical Methods

10-10:50 AM MWF in Olin 103

Notes on Lecture 27 – Chap. 9 in F & W

Introduction to hydrodynamics

- 1. Motivation for topic**
- 2. Newton's laws for fluids**
- 3. Conservation relations**

21	Mon, 10/16/2023	Chap. 7	The wave and other partial differential equations	#15
22	Wed, 10/18/2023	Chap. 7	Sturm-Liouville equations	#16
23	Fri, 10/20/2023	Chap. 7	Sturm-Liouville equations	#17
24	Mon, 10/23/2023	Chap. 7	Laplace transforms and complex functions	#18
25	Wed, 10/25/2023	Chap. 7	Complex integration	#19
26	Fri, 10/27/2023	Chap. 8	Wave motion in 2 dimensional membranes	#20
27	Mon, 10/30/2023	Chap. 9	Motion in 3 dimensional ideal fluids	#21
28	Wed, 11/01/2023	Chap. 9	Motion in 3 dimensional ideal fluids	
29	Fri, 11/03/2023			
31	Mon, 11/06/2023			
32	Wed, 11/08/2023			
33	Fri, 11/10/2023			
34	Mon, 11/13/2023			
35	Wed, 11/15/2023			
36	Fri, 11/17/2023			
37	Mon, 11/20/2023			
	Wed, 11/22/2023	Thanksgiving		

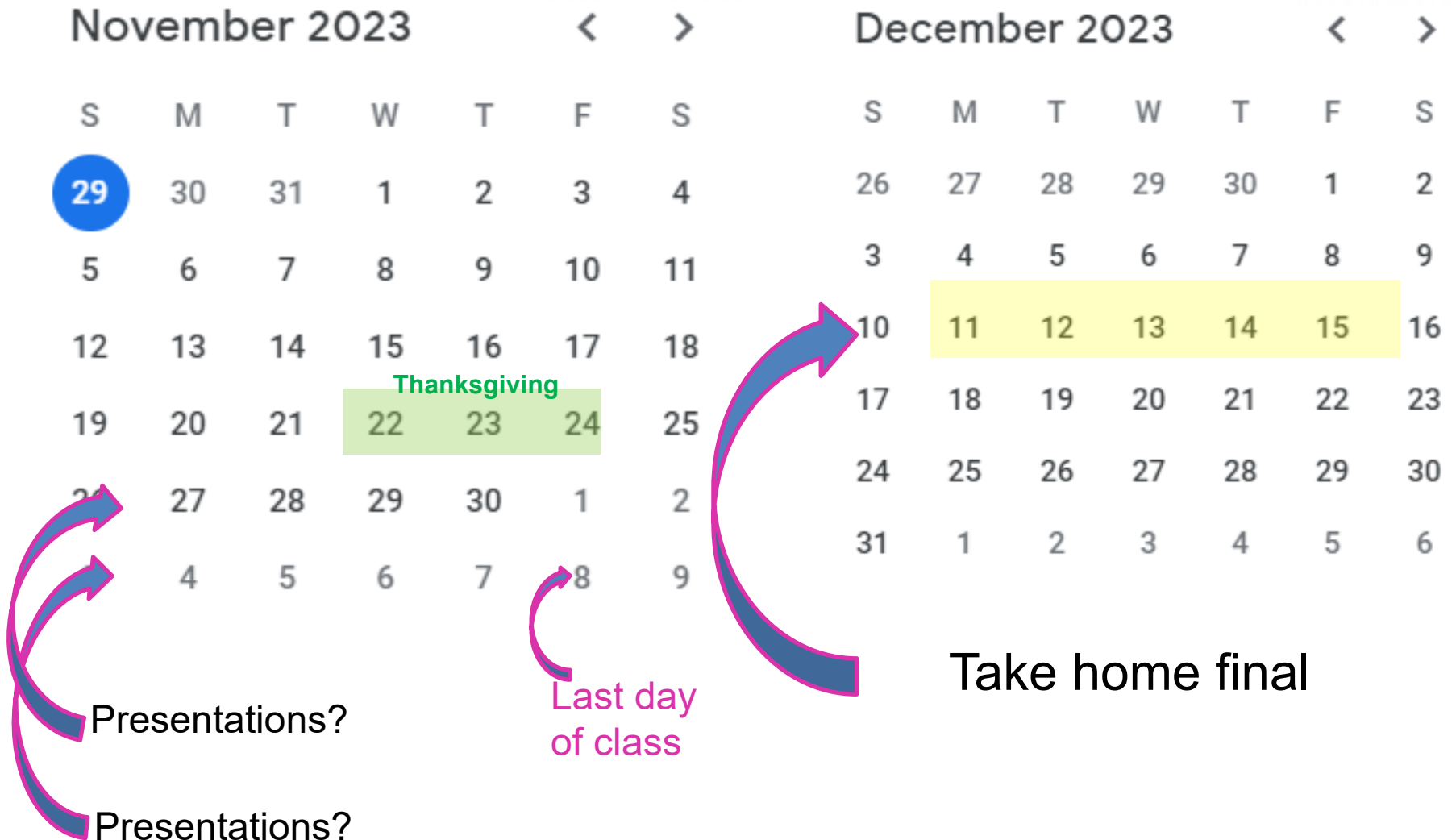
PHY 711 -- Assignment #21

Assigned: 10/30/2023 Due: 11/06/2023

Read Chapter 9 in **Fetter & Walecka**.

1. A tank having an area of 100 m^2 is open to the atmosphere and contains 1000 m^3 of water. It has a spigot at its bottom which has a height of 1 m above a drain in the floor. A hose having a diameter of 1 cm is used to empty the water from the tank when the spigot is opened. Using Bernoulli's analysis, estimate the time it takes to empty the tank via the floor drain.

Now is a good time to start thinking about your projects --



Presentation expectations

- Prepare with powerpoint (or equivalent) for ~ 10 minutes and expect ~ 5 minutes for discussion (3 presentations per day)
- To accommodate all students, we will need 3 days....
- Details listed on webpage

<http://users.wfu.edu/natalie/f23phy711/info/computational.html>

Project

The purpose of this assignment is to provide an opportunity for you to study a topic of your choice in greater depth. The general guideline for your choice of project is that it should have something to do with classical mechanics, and there should be some degree of analytic or numerical computation associated with the project. The completed project will include a short write-up and a presentation to the class. You may design your own project or use one of the following list (which will be updated throughout the term).

- Explain the details of a homework problem that was assigned or one you design, including the basic principles and the solution methods and results.
- Consider a scattering experiment in which you specify the spherically symmetric interaction potential $V(r)$. Write a computer program (using your favorite language) to evaluate the scattering cross section for your system. (Depending on your choice, you may wish to present your results either in the center-of-mass or lab frames of reference.)
- Consider the Foucault Pendulum. Analyze the equations of motion including both the horizontal and vertical motions. You can either solve the equations exactly or use perturbation theory. Compare the effects of the vertical motion to the effects of air friction.
- Consider a model system of 2 or more interacting particles with appropriate initial conditions, using numerical methods to find out how the system evolves in time and space. For few particles and special initial conditions this approach can be used to explore orbital mechanics. For many particles and random initial conditions, this approach can be used to explore statistical mechanics via molecular dynamics simulations.
- Examine the normal modes of vibration for a model system with 3 or more masses in 2 or 3 dimensions.
- Analyze the soliton equations beyond what was covered in class.



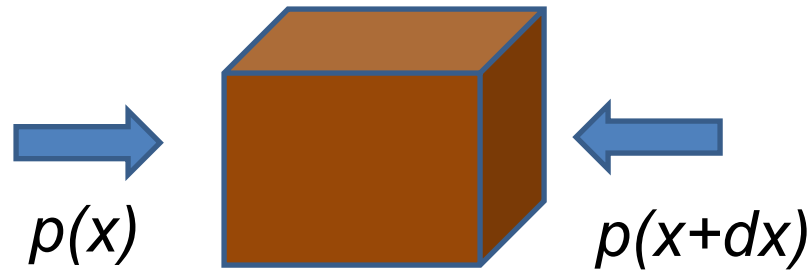
Hydrodynamic analysis

Motivation

1. Natural progression from strings, membranes, fluids; description of 1, 2, and 3 dimensional continua
2. Interesting and technologically important phenomena associated with fluids

Plan

1. Newton's laws for fluids (leaving out dissipative effects for now)
2. Continuity equation
3. Stress tensor
4. Energy relations
5. Bernoulli's theorem
6. Various examples
7. Sound waves



$$\begin{aligned} F_{\text{pressure}} \Big|_x &= \left(-p(x+dx, y, z) + p(x, y, z) \right) dydz \\ &= \frac{\left(-p(x+dx, y, z) + p(x, y, z) \right)}{dx} dx dy dz \\ &= -\frac{\partial p}{\partial x} dV \end{aligned}$$



Newton's equations for fluids -- continued

$$m\mathbf{a} = \mathbf{F}_{\text{applied}} + \mathbf{F}_{\text{pressure}}$$

$$\rho dV \frac{d\mathbf{v}}{dt} = \mathbf{f}_{\text{applied}} \rho dV - (\nabla p) dV$$

$$\rho \frac{d\mathbf{v}}{dt} = \rho \mathbf{f}_{\text{applied}} - \nabla p$$

$$m = \rho dV$$

$$\mathbf{f}_{\text{applied}} = \frac{\mathbf{F}_{\text{applied}}}{m}$$

$$\mathbf{F}_{\text{pressure}} = -\nabla p dV$$

Detailed analysis of acceleration term:

$$\mathbf{v} = \mathbf{v}(x, y, z, t)$$

$$\frac{d\mathbf{v}}{dt} = \frac{\partial \mathbf{v}}{\partial x} \frac{dx}{dt} + \frac{\partial \mathbf{v}}{\partial y} \frac{dy}{dt} + \frac{\partial \mathbf{v}}{\partial z} \frac{dz}{dt} + \frac{\partial \mathbf{v}}{\partial t}$$

$$\frac{d\mathbf{v}}{dt} = \frac{\partial \mathbf{v}}{\partial x} v_x + \frac{\partial \mathbf{v}}{\partial y} v_y + \frac{\partial \mathbf{v}}{\partial z} v_z + \frac{\partial \mathbf{v}}{\partial t}$$

$$\frac{d\mathbf{v}}{dt} = (\mathbf{v} \cdot \nabla) \mathbf{v} + \frac{\partial \mathbf{v}}{\partial t}$$

Note that : $\mathbf{v} \equiv v_x \hat{\mathbf{x}} + v_y \hat{\mathbf{y}} + v_z \hat{\mathbf{z}}$

$$\frac{\partial \mathbf{v}}{\partial x} v_x + \frac{\partial \mathbf{v}}{\partial y} v_y + \frac{\partial \mathbf{v}}{\partial z} v_z = \nabla \left(\frac{1}{2} \mathbf{v} \cdot \mathbf{v} \right) - \mathbf{v} \times (\nabla \times \mathbf{v})$$



Newton's equations for fluids -- continued

$$\rho \frac{d\mathbf{v}}{dt} = \rho \left((\mathbf{v} \cdot \nabla) \mathbf{v} + \frac{\partial \mathbf{v}}{\partial t} \right) = \rho \mathbf{f}_{\text{applied}} - \nabla p$$

$$\rho \left(\nabla \left(\frac{1}{2} \mathbf{v} \cdot \mathbf{v} \right) - \mathbf{v} \times (\nabla \times \mathbf{v}) + \frac{\partial \mathbf{v}}{\partial t} \right) = \rho \mathbf{f}_{\text{applied}} - \nabla p$$

$$\frac{\partial \mathbf{v}}{\partial t} + \nabla \left(\frac{1}{2} v^2 \right) - \mathbf{v} \times (\nabla \times \mathbf{v}) = \mathbf{f}_{\text{applied}} - \frac{\nabla p}{\rho}$$

Detail – What is irrotational flow?

Irrotational flow: $\nabla \times \mathbf{v} = 0$

$$\nabla \times \mathbf{v} = \hat{\mathbf{x}} \left(\frac{\partial v_z}{\partial y} - \frac{\partial v_y}{\partial z} \right) + \hat{\mathbf{y}} \left(\frac{\partial v_x}{\partial z} - \frac{\partial v_z}{\partial x} \right) + \hat{\mathbf{z}} \left(\frac{\partial v_y}{\partial x} - \frac{\partial v_x}{\partial y} \right)$$

Which of the following vector functions have zero curl?

- $\mathbf{v} = C\hat{\mathbf{x}}$ (C is a constant)
- $\mathbf{v} = Cx\hat{\mathbf{x}}$
- $\mathbf{v} = Cy\hat{\mathbf{x}}$

Solution of Euler's equation for fluids

$$\frac{\partial \mathbf{v}}{\partial t} + \nabla \left(\frac{1}{2} v^2 \right) - \mathbf{v} \times (\nabla \times \mathbf{v}) = \mathbf{f}_{\text{applied}} - \frac{\nabla p}{\rho}$$

Consider the following restrictions:

1. $(\nabla \times \mathbf{v}) = 0$ "irrotational flow"

$$\Rightarrow \mathbf{v} = -\nabla \Phi \quad \Phi \text{ is "velocity potential"}$$

2. $\mathbf{f}_{\text{applied}} = -\nabla U$ conservative applied force

3. $\rho = (\text{constant})$ incompressible fluid

$$\frac{\partial(-\nabla \Phi)}{\partial t} + \nabla \left(\frac{1}{2} v^2 \right) = -\nabla U - \frac{\nabla p}{\rho}$$

$$\Rightarrow \nabla \left(\frac{p}{\rho} + U + \frac{1}{2} v^2 - \frac{\partial \Phi}{\partial t} \right) = 0$$

Bernoulli's integral of Euler's equation for irrotational and incompressible fluid

$$\nabla \left(\frac{p}{\rho} + U + \frac{1}{2} v^2 - \frac{\partial \Phi}{\partial t} \right) = 0$$

Integrating over space :

$$\frac{p}{\rho} + U + \frac{1}{2} v^2 - \frac{\partial \Phi}{\partial t} = C(t)$$

where $\mathbf{v} = -\nabla \Phi(\mathbf{r}, t) = -\nabla (\Phi(\mathbf{r}, t) + C'(t))$

$$\Rightarrow \frac{p}{\rho} + U + \frac{1}{2} v^2 - \frac{\partial \Phi}{\partial t} = 0 \quad \text{Bernoulli's theorem}$$

Examples of Bernoulli's theorem

$$\frac{p}{\rho} + U + \frac{1}{2} v^2 - \frac{\partial \Phi}{\partial t} = 0$$

Modified form; assuming $\frac{\partial \Phi}{\partial t} = 0$

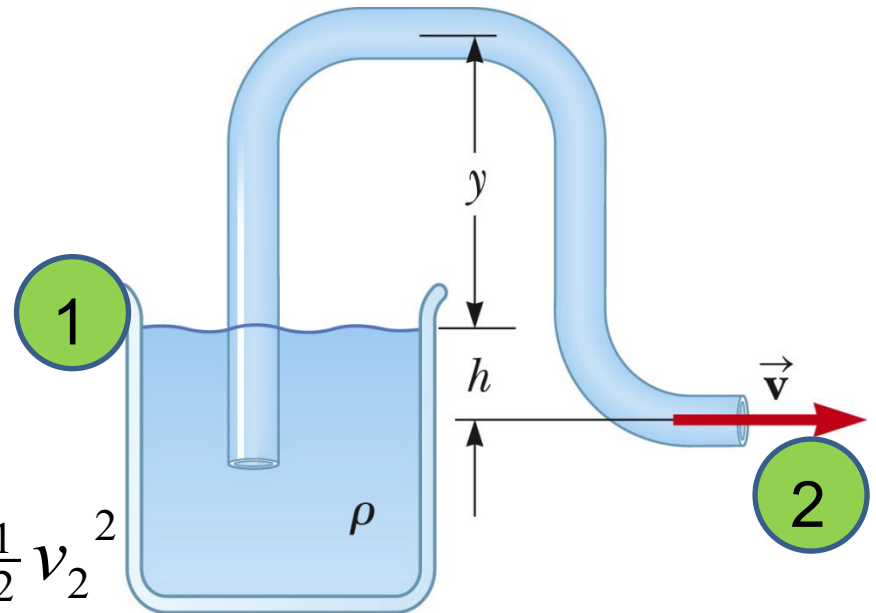
$$\frac{p}{\rho} + U + \frac{1}{2} v^2 = \text{constant}$$

$$p_1 = p_2 = p_{atm}$$

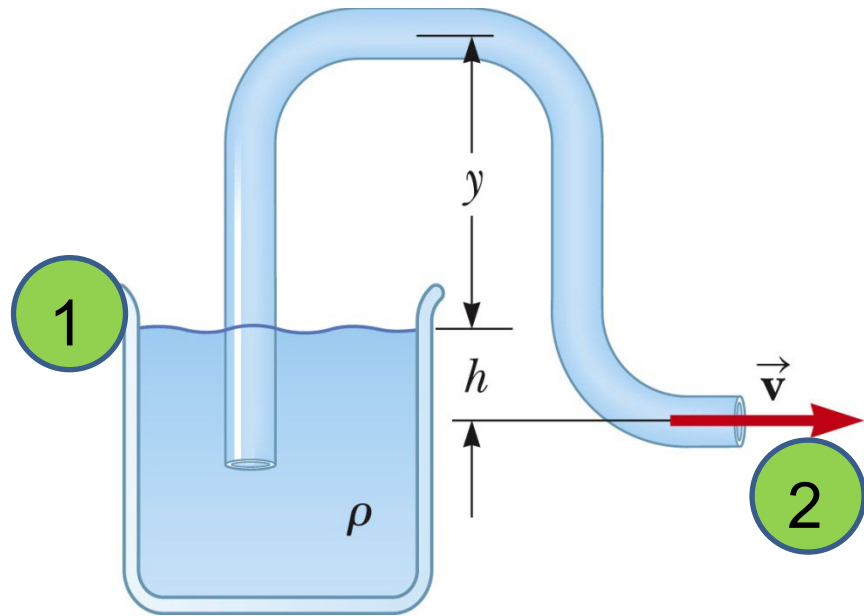
$$U_1 - U_2 = gh$$

$$v_1 \approx 0$$

$$\frac{p_1}{\rho} + U_1 + \frac{1}{2} v_1^2 = \frac{p_2}{\rho} + U_2 + \frac{1}{2} v_2^2$$



Examples of Bernoulli's theorem -- continued



$$p_1 = p_2 = p_{atm}$$

$$U_1 - U_2 = gh$$

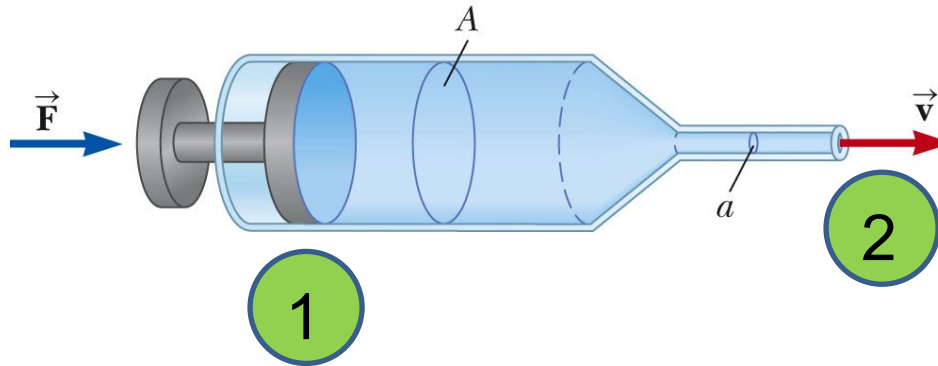
$$v_1 \approx 0$$

$$\frac{p_1}{\rho} + U_1 + \frac{1}{2} v_1^2 = \frac{p_2}{\rho} + U_2 + \frac{1}{2} v_2^2$$

$$v_2 \approx \sqrt{2gh}$$

Examples of Bernoulli's theorem -- continued

$$\frac{p}{\rho} + U + \frac{1}{2} v^2 = \text{constant}$$



$$p_1 = \frac{F}{A} + p_{atm}$$

$$p_2 = p_{atm}$$

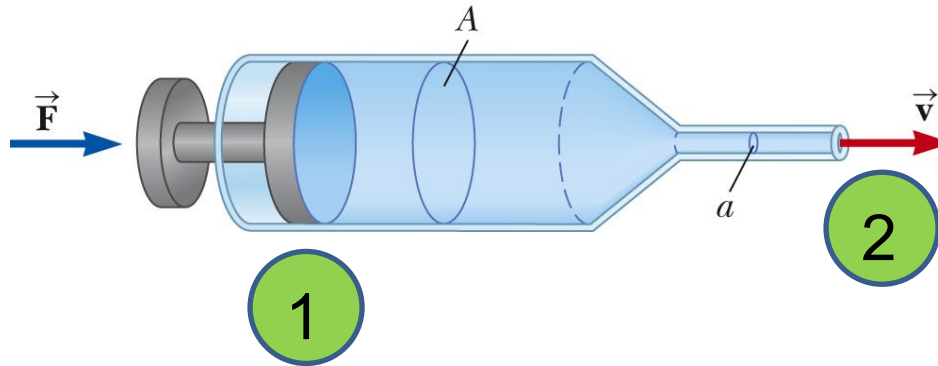
$$U_1 = U_2$$

$$v_1 A = v_2 a \quad \text{continuity equation}$$

$$\frac{p_1}{\rho} + U_1 + \frac{1}{2} v_1^2 = \frac{p_2}{\rho} + U_2 + \frac{1}{2} v_2^2$$

Examples of Bernoulli's theorem -- continued

$$\frac{p}{\rho} + U + \frac{1}{2} v^2 = \text{constant}$$



$$\frac{2F}{A} = v_2^2 \left(1 - \left(\frac{a}{A} \right)^2 \right)$$

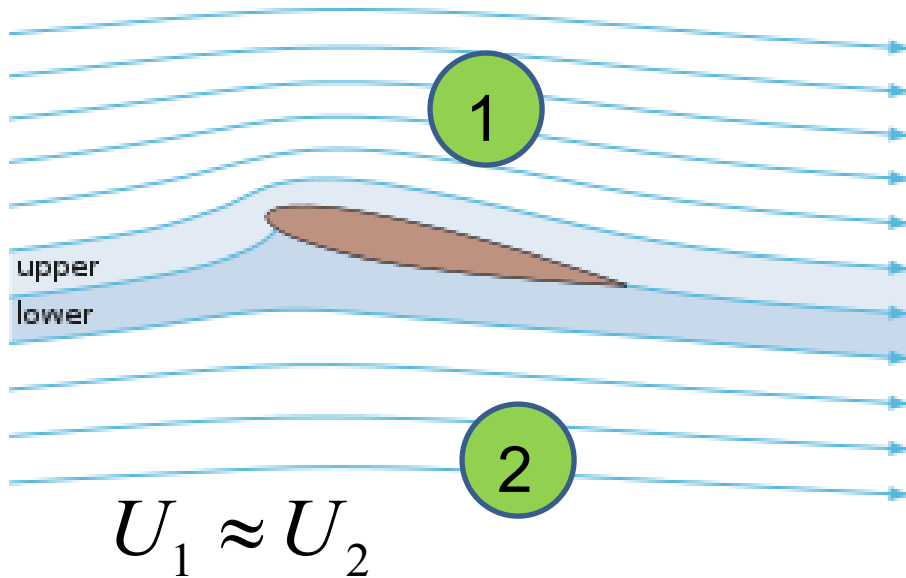
$$v_2 = \sqrt{\frac{2F / A}{1 - \left(\frac{a}{A} \right)^2}}$$

Examples of Bernoulli's theorem – continued

Approximate explanation of airplane lift

Cross section view of airplane wing

http://en.wikipedia.org/wiki/Lift_%28force%29



$$\frac{p_1}{\rho} + U_1 + \frac{1}{2}v_1^2 = \frac{p_2}{\rho} + U_2 + \frac{1}{2}v_2^2$$

$$p_2 - p_1 = \frac{1}{2}\rho(v_1^2 - v_2^2)$$

Question -- What aspects do over simplified Bernoulli's equation not include in studying fluid dynamics?

According to a Scientific American article, the conclusion that $v_2 > v_1$ because of the shape of the airplane wing is not quite true and because viscosity has an important effect. Numerical modeling reveal a more complicated picture.

<https://www.scientificamerican.com/article/no-one-can-explain-why-planes-stay-in-the-air/>



At NASA Ames Fluid Mechanics Laboratory, streamlines of dye in a water channel interact with a model airplane. Credit: *Ian Allen* (copied from Scientific American page mentioned above).

Continuity equation connecting fluid density and velocity:

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

$$\frac{\partial \rho}{\partial t} + \rho (\nabla \cdot \mathbf{v}) + (\nabla \rho) \cdot \mathbf{v} = 0$$

Consider: $\frac{d\rho}{dt} = \frac{\partial \rho}{\partial t} + (\nabla \rho) \cdot \mathbf{v}$

$$\Rightarrow \frac{d\rho}{dt} + \rho (\nabla \cdot \mathbf{v}) = 0$$

alternative form

of continuity equation



Some details on the velocity potential

Continuity equation :

$$\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{v}) = 0$$

$$\frac{\partial \rho}{\partial t} + \rho(\nabla \cdot \mathbf{v}) + (\nabla \rho) \cdot \mathbf{v} = 0$$

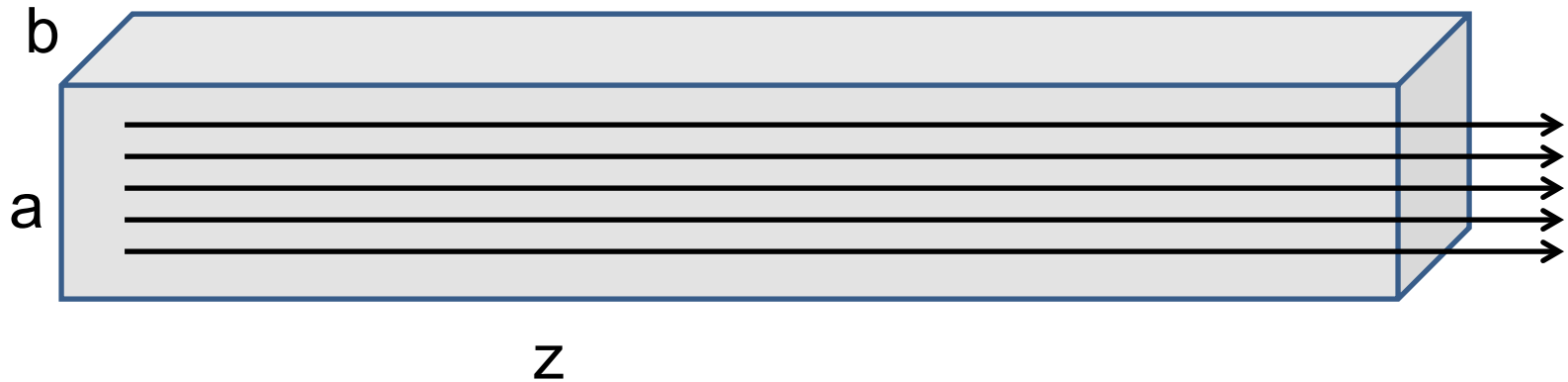
For incompressible fluid : $\rho = (\text{constant})$

$$\Rightarrow \nabla \cdot \mathbf{v} = 0$$

Irrotational flow : $\nabla \times \mathbf{v} = 0 \qquad \Rightarrow \mathbf{v} = -\nabla \Phi$

$$\Rightarrow \nabla^2 \Phi = 0$$

Example – uniform flow



$$\nabla^2 \Phi = 0$$

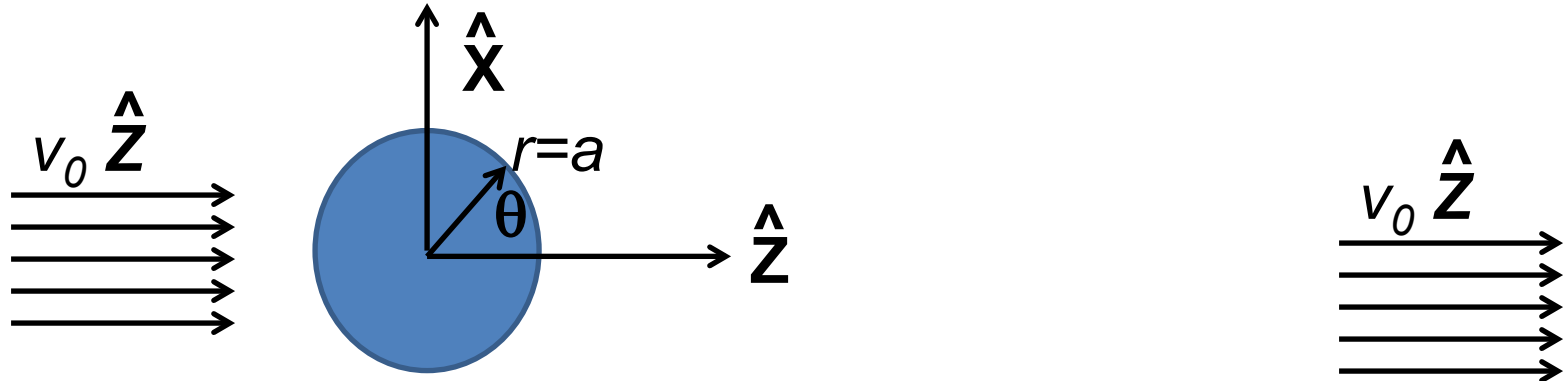
$$\frac{\partial^2 \Phi}{\partial x^2} + \frac{\partial^2 \Phi}{\partial y^2} + \frac{\partial^2 \Phi}{\partial z^2} = 0$$

Possible solution :

$$\Phi = -v_o z$$

$$\mathbf{v} = -\nabla \Phi = v_o \hat{\mathbf{z}}$$

Example – flow around a long cylinder (oriented in the Y direction)



$$\nabla^2 \Phi = 0$$

$$\left. \frac{\partial \Phi}{\partial r} \right|_{r=a} = 0$$

Laplace equation in cylindrical coordinates

(r, θ , defined in x - z plane; y representing cylinder axis)

$$\nabla^2 \Phi = 0 = \frac{1}{r} \frac{\partial}{\partial r} r \frac{\partial \Phi}{\partial r} + \frac{1}{r^2} \frac{\partial^2 \Phi}{\partial \theta^2} + \frac{\partial^2 \Phi}{\partial y^2}$$

In our case, there is no motion in the y dimension

$$\Rightarrow \Phi(r, \theta, y) = \Phi(r, \theta)$$

From boundary condition : $v_z(r \rightarrow \infty) = v_0$

$$\frac{\partial \Phi}{\partial z}(r \rightarrow \infty) = -v_0 \quad \Rightarrow \quad \Phi(r \rightarrow \infty, \theta) = -v_0 r \cos \theta$$

Note that : $\frac{\partial^2 \cos \theta}{\partial \theta^2} = -\cos \theta$

Guess form : $\Phi(r, \theta) = f(r) \cos \theta$

Necessary equation for radial function

$$\frac{1}{r} \frac{\partial}{\partial r} r \frac{\partial f}{\partial r} - \frac{1}{r^2} f = 0$$

$$f(r) = Ar + \frac{B}{r} \quad \text{where } A, B \text{ are constants}$$


Boundary condition on cylinder surface :

$$\left. \frac{\partial \Phi}{\partial r} \right|_{r=a} = 0$$

$$\left. \frac{df}{dr} \right|_{(r=a)} = 0 = A - \frac{B}{a^2}$$

$$\Rightarrow B = Aa^2$$

Boundary condition at ∞ : $\Rightarrow A = -v_0$


$$\Phi(r, \theta) = -v_0 \left(r + \frac{a^2}{r} \right) \cos \theta$$

$$v_r = -\frac{\partial \Phi}{\partial r} = v_0 \left(1 - \frac{a^2}{r^2} \right) \cos \theta$$

$$v_\theta = -\frac{1}{r} \frac{\partial \Phi}{\partial \theta} = v_0 \left(1 + \frac{a^2}{r^2} \right) \sin \theta$$

For 3-dimensional system, consider a spherical obstruction

Laplacian in spherical polar coordinates:

$$\nabla^2 \Phi = 0 = \frac{1}{r^2} \frac{\partial}{\partial r} \left(r^2 \frac{\partial \Phi}{\partial r} \right) + \frac{1}{r^2 \sin \theta} \frac{\partial}{\partial \theta} \left(\sin \theta \frac{\partial \Phi}{\partial \theta} \right) + \frac{1}{r^2 \sin^2 \theta} \frac{\partial^2 \Phi}{\partial \varphi^2}$$

to be continued ...